

Description

The present invention relates to a hoist and traction machine, and more particularly to a hoist and traction machine provided with; a load sheave; a driving shaft provided with driven member and for driving the load sheave; a driving member screwable with the driving shaft; a braking pawl and a braking ratchet wheel and braking plates, which are interposed between the driving member and the driven member and constitute a mechanical brake; and driving means, such as a manual lever or the like, for driving the driving member normally or reversely.

An example of a hoist and traction machine is disclosed in German Patent Specification No. DE-A-3323150. This machine comprises a load sheave, a driving shaft, having a driven member for driving the load sheave, a driving member threadedly mounted on said driving shaft, a braking ratchet wheel engageable with a braking pawl, a mechanical brake comprising braking plates interposed between said driving member and said driven member, driving means for normally and reversely driving the driving member and a cover for housing the said mechanical brake so that the driving member is not in contact with the cover. The hoist and traction machine further comprises a free-rotation control device for controlling operation of the mechanical brake so as to provide free rotation for the load sheave. The free rotation control device comprises a stop means at an axial end of the driving shaft so as to be non-rotatable and axially immovable with respect to the driving shaft and a free-rotation operating handle provided between the stop means and the driving member in such a manner as to be axially movable between a first position, in proximity to said driving member, and a second position, spaced therefrom. The machine also includes regulation means, for regulating a range of relative rotation of the driving member with respect of the driving shaft when the handle is put in the first position.

An object of the present invention is to provide a hoist and traction machine which can enlarge an input range of a pulling force of the chain during the free rotation control, hold a load sheave in the free rotation state without requiring skill, perform quick pulling work of the chain, perform with ease the free rotation control, and incapacitate the free rotation control when subject to a load, thereby providing a high level of safety.

Another object of the present invention is to provide a hoist and traction machine which can prevent an over-load besides the above-mentioned free rotation control and use an adjusting member for adjusting a slip load at an overload preventing mechanism also as a member for holding the free

rotation operation at the free rotation control, thereby enabling economy in the number of parts and enabling free rotation control and overload prevention to be performed.

The present invention provides a hoisting and traction machine comprising a load sheave; a driving shaft, having a driven member, for driving said load sheave; a driving member threadingly mounted to said driving shaft; a braking pawl; a braking ratchet wheel engageable with said braking pawl; braking plates interposed between said driving member and said driven member, constituting a mechanical brake; driving means for normally and reversely driving said driving member; a cover for covering said mechanical brake, wherein said driving member does not contact said cover, said hoist and traction machine further comprising a free rotation control device for controlling operation of said mechanical brake so as to provide free rotation for said load sheave, said free rotation control device comprising:

(a) a stopper provided at an axial end of said driving shaft so as to be non-rotatable and axially immovable with respect to said driving shaft;

(b) a free-rotation operation handle provided between said stopper and said driving member in such a manner as to be axially movable between a first position, in proximity to said driving member, and a second position, apart therefrom;

(c) regulation means, for regulating a relative rotation range of said driving member with respect to said driving shaft when said handle is put in the first position, characterized in that

(i) an elastically biasing member is interposed between said stopper and said handle, for biasing said handle toward said first position in proximity to said driving member,

(ii) said regulation means is provided between said handle and said driving member so that, when said handle is put in the second position, the regulation is released;

(iii) free-rotation-control holding means is provided between said handle and the driving member for applying a biasing force to said driving member from said elastically biasing member, when said handle is put in the second position to release said regulation and is rotated so as to perform the free rotation operation, so as to hold said free rotation operation; and

(iv) said handle is non-rotatable relative to said driving shaft due to engagement with said stopper, said handle being rotationally operable to rotate said driving shaft to cause said driving member to screw backwards in the non-braking direction, so as to provide

the free rotation operation.

The above-mentioned construction provides the following operational effect. In detail, the operating handle is moved against the elastic biasing member toward the second position where the operating handle moves away from the driving member so as to release the regulation by the regulation means and to enable the handle to rotate normally, whereby the handle rotates to forcibly rotate the driving member so as to enable the driving member to move away from the braking plate. Accordingly, at first, it is possible to release the braking action of a mechanical brake comprising a braking ratchet wheel and braking plates. Then, the free rotation control holding means applies a biasing force of the elastic biasing member onto the driving member so as to hold the state where the braking action by the brake is released, that is, the state of free rotation. Accordingly, an input range of the pulling force of chain during the free rotation control is enlarged by the holding effect, thereby enabling the chain at the load side to be quickly elongated and shortened without requiring skill. Moreover, the operating handle, which is operated to put the load sheave in the free rotation state, can increase its ratio of radius of movement in comparison with the case where the driving shaft is directly rotated, thereby enabling the free rotation to be performed by a light force to that extent.

Accordingly, the chain can simply be elongated or shortened toward the load side without requiring skill.

When the chain engaged with the load sheave is subjected to the load, the operating handle, even when operated for free rotation, reversely rotates by the load with respect to the driving member so as not to freely rotate the load sheave, thereby improving safety.

In one embodiment of the present invention the regulation means and free rotation control means are so constructed that the regulation means is provided with a pair of projecting portions each having regulating surfaces for regulating a relative-rotation range of the driving member with respect to the driving shaft and with engaging projections fitted between the projecting portions to engage with the regulation surfaces respectively when the operating handle is put in the first position, and the free rotation control holding means is provided with free rotation control surfaces positioned out of the regulation range by the regulating surfaces, so that when the operating handle is put in the second position to freely rotate the load sheave, the engaging projections are adapted to come in elastic contact with the free rotation regulating surfaces respectively.

Furthermore, in the above-mentioned construction, it is preferable that regulation portions for

regulating the free rotation operation range by the operating handle are provided at the front in the free rotation operation direction of the operating handle.

In this case, when the driving shaft is rotated to disengage the driving member from the braking plate at the mechanical brake during the free rotation operation by the operating handle, the driving member can be regulated for relative rotation thereof with respect to the driving shaft, whereby to avoid the situation in which, when the chain is drawn out by being pulled at the no load side, the free rotation cannot be released, even when too much is pulled out. In other words, when too much chain is pulled out, a stopper provided at the no load side of the chain abuts against the frame of the hoist and traction machine so as to restrain the chain from being further pulled out. Hence, when the chain is quickly pulled out and the stopper abuts against the frame so as to suddenly stop the rotation of driving shaft, the driving member rotates by its force of inertia in spite of stopping the driving shaft, whereby the driving member moves further away from the braking plate and the projections more intensively abut against the free rotation control surfaces to result in incapacitation of release of free rotation, but such a problem can be solved by the above-mentioned regulation portions.

In the above-mentioned construction, it is preferable that the driving shaft has a positioning portion for setting the first position for the operating handle, the first position being set in the position where the operating handle is in non-contact with the driving member to be discussed below.

In an example of an arrangement according to the present invention there is provided an overload prevention mechanism as follows:

Namely, the driving member comprising a first driving member having a boss screwable with the driving shaft and a larger diameter portion opposite to the brake plate at the mechanical brake and a second driving member supported to the boss of the first driving member relative-rotatably. Onto the boss of the first driving member are supported friction plates and an elastic member and there is screwably attached an adjusting member for changing a biasing force applied by the elastic member to the friction plates so as to adjust a slip load, the adjusting member being opposite to the operating handle and provided with a regulation portion for regulating the relative rotation range of the driving member with respect to the driving shaft in the first position of the operating handle and with free rotation control surfaces against which the driving handle elastically abuts so as to hold the free rotation operation of the driving shaft by the handle.

In this construction, the free rotation operation of operating handle can freely and rotatably control the load sheave as above-mentioned and can hold the free rotation operation, so that, when the operating handle is operated so as not to freely rotate the load sheave, the first driving member is screwed forwardly and backwardly with respect to the driven member to actuate the mechanical brake, and the overload prevention mechanism adjustable of the rating load by the adjusting member can be operated.

Accordingly, the overload prevention mechanism is operated to prevent overloading and also the driving shaft can be kept in the free rotation state by the free rotation operating handle without requiring skill. Moreover, the adjusting member for adjusting the slip load onto the overload prevention mechanism can be used both as parts for adjusting the rating load of overload prevention mechanism and holding the driving shaft in the free rotation state, thereby enabling the number of parts to be saved.

Also, it is preferable that the hoist and traction machine provided with the overload prevention mechanism has the following construction :

The regulation portions of the adjusting member each comprise a cutout having a pair of regulating surfaces for regulating the relative rotation range of the driving member with respect to the driving shaft, the operating handle being provided with engaging projections each entering into the cutout in the first position of the operating handle to engage with the regulating surface and coming in elastic contact with the free rotation control surface in the second position.

Further, the overload prevention mechanism may be so constructed that between the boss of the first driving member and the second driving member is provided a unidirectional rotation mechanism which makes the second driving member, when rotating in the driving direction, freely rotatable with respect to the first driving member and which makes the second driving member, when rotating in the not-driving direction, integrally rotatable with the first driving member, the unidirectional rotation mechanism comprising an engaging member held to be forwardly or backwardly movable to one of the first and second driving members and an engaging groove engageable with the engaging member when rotating in the engaging direction thereof during the rotation of the second driving member in the not-driving direction, the engaging groove being provided plurally and circumferentially.

In addition, the hoist and traction machine provided with the overload prevention mechanism uses the adjusting member also as a member for holding the free rotation by the operating handle, in

which the free rotation is held by bringing the projections at the operating handle in elastic contact with the free rotation control surfaces of the adjusting member, whereby there is no fear that the slip load set by the adjusting member changes by the above-mentioned holding.

These and other objects of the invention will become more apparent in the detailed description and examples which follow.

In the drawings:-

Fig.1 is a longitudinally sectional view of a first embodiment of a lever type hoist and traction machine of the invention,

Fig. 2 is an illustration of the engaging state of ridges 29 at an operating handle with engaging grooves 30 at a stopper 17,

Fig.3 is a front view of a driving member, in which a relative rotation range of the driving member with respect to a driving shaft and a rotary position of each engaging projection with respect to the driving member during the free rotation are shown,

Fig.4 is a sectional view taken on the line A-A in Fig.3,

Fig.5 is a longitudinally sectional view of the state where the hoist and traction machine is operated to freely rotate and the free rotation control is held,

Fig.6 is a longitudinally sectional view of a second embodiment of the lever type hoist and traction machine of the invention,

Fig.7 is a sectional view of the principal portion of the invention, showing the state where the machine is freely rotated and the free rotation control is held, corresponding to Fig.6.

Fig. 8 is a sectional view taken on the line B-B in Fig. 7, and

Fig. 9 is an illustration of a unidirectional rotation mechanism provided between a first driving member and a second driving member.

Firstly, the first embodiment of the hoist and traction machine shown in Figs. 1 through 5 will be described.

The first embodiment of the lever type hoist and traction machine, as shown in Fig. 1, is so constructed that a tubular shaft 4 having a load sheave 3 is rotatably supported between a first side plate 1 and a second side plate 2 disposed opposite to each other and spaced at a predetermined interval, in the tubular shaft 4 is relatively rotatably supported a driving shaft 5 to which a driving torque is transmitted from an operating lever to be discussed below, and a reduction gear mechanism 6 comprising a plurality of reducing gears is interposed between an outside end of a driving shaft 5 projecting from the second side plate 2 and the load sheave 3, so that the reduction gear mechanism 6 reduces the driving torque and

transmits it toward the load sheave 3.

A driven member 7 comprising a hub having a flange screws with an outer portion of the driving shaft 5 projecting from the first side plate 1, a driving member 8 having at the outer periphery thereof teeth 8a screws with the driving shaft 5 at the outside of the driven member 7, a pair of braking plates 9 and 10 and a braking ratchet wheel 11 are interposed between the driving member 8 and the driven member 7, and a braking pawl 12 engageable with the braking ratchet wheel 11 is provided at the first side plate 1, so that the braking ratchet wheel 11 and braking plates 9 and 10 constitute a mechanical brake 13.

Furthermore, outside a brake cover 13a for covering the outer periphery of the mechanical brake 13 and radially outwardly of the driving member 8 is provided driven means comprising an operating lever 16 which has a pawl member 14 provided with normal and reverse rotation pawls engageable with the teeth 8a provided at the outer periphery of the driving member 8 and has a control portion 15 for controlling the pawl member 14 to engage with or disengage from the teeth 8a.

In the lever type hoist and traction machine constructed as above-mentioned, a stopper 17 is provided at an axial end of the driving shaft 5, an operating handle 18 not-relative-rotatable with respect to the driving shaft 5 is interposed between the stopper 17 and the driving member 8 in relation of being axially movable across from a first position where the handle 18 moves toward the driving member 8 to a second position where the same moves away therefrom, between the operating handle 18 and the stopper 17 is provided an elastic biasing member 19 mainly comprising a coil spring and for biasing the handle 18 toward the driving member 8, and between the operating handle 18 and the driving member 8 is provided regulation means which can regulate a relative rotation range of the driving member 8 with respect to the driving shaft 5 and release the regulation of the range by moving the handle 18 away from the driving member 8.

In other words, in the first embodiment shown in Fig.1, first and second threaded portion 20 and 21 and a serration portion 23 are provided on the driving shaft 5, the driven member 7 screws with the first threaded portion 20 and the driving member 8 with the second threaded portion 21, a coil spring 24 is interposed between the driven member 7 and the driving member 8 and restricts the axial movement of driven member 7 with respect to the driving shaft 5, and the driving member 8 is normally rotated with respect to the driving shaft 5 so as to screw forward in the leftward direction in Fig.1. Also, a pair of sleeves 25 and 26 are fitted onto the serration portion 23 on the driving shaft 5

axially outside of the driving member 8, a flange 25a is provided at the first sleeve 25, a stopper 17 is mounted by serration coupling to the end of the serration portion 23 outside the second sleeve 26, and a nut 27 is tightened to fix the stopper 17 to the driving shaft 5 through the sleeves 25 and 26.

Onto the second sleeve 26 is fitted a bore 28a provided at a boss 28 of the operating handle 18, so that the operating handle 18 is interposed between the stopper 17 and the driving member 8 and, as shown in Fig.2, a pair of ridges 29 are provided at the inner periphery of operating handle 18 so as to engage with engaging grooves 30 provided at the outer periphery of stopper 17 as shown in Figs.1 and 2, thereby making the operating handle 18 not-relative-rotatable with respect to the driving shaft 5.

Between the axially outside surface of the boss 28 of operating handle 18 and the axially inside surface of the stopper 17 opposite to the boss 28 is interposed the elastic biasing member 19 in contact with the respective side surfaces so as to bias the operating handle 18 toward the flange 25a of the first sleeve 25 in the direction of moving away from the stopper 17, in other words, toward the driving member 8.

Furthermore, two engaging projections 31 projecting toward the driving member 8, are, as shown in the dotted lines in Fig.3, symmetrically provided at the radial end portion at the rear surface of the boss 28 of operating handle 18, a pair of projecting portions 32, as shown in Figs.3 and 4, are symmetrically provided at one axial side of the driving member 8 opposite to the boss 28 at the operating handle 18, at the projecting side surfaces of the projecting portions 32 are provided first and second regulating surfaces 33 and 34 which, when the operating handle 18 is rotated not to axially move away from the driving member 8 but relative-rotate the driving member 8 with respect to the driving shaft 5, engage with the engaging projections 31 respectively to regulate the relative rotation range of driving member 8 with respect to the driving shaft 5, and at the projecting front surfaces of the projecting portions 32 are provided free rotation control surfaces 35 which, when the operating handle 18 is moved away from the driving member 8 and relative-rotated with respect thereto, are biased by the elastic biasing member 19, so as to be contactable with the utmost ends of engaging projections 31 respectively. Furthermore, at the projecting front surfaces of the projecting portions 32 are provided regulating portions 36 which rise from the free rotation control surfaces 35 and which, when driving member 8 relative-rotates with respect to the driving shaft 5 in the state where the utmost end faces of engaging projections 31 contact with the free rotation control surfaces 35 respectively, en-

gage with the front sides of the engaging projections 31 in the rotating direction thereof respectively.

In the above construction the driven member 7 and driving member 8 screw with the first and second threaded portions 20 and 21 on the driving shaft 5 in consideration of the workability and strength thereof, in which the first threaded portion 20 may be serrations. Also, the coil spring 24 restricts forward screwing of the driven member 7 by the driven member 7, but a snap ring, such as an E-ring, may be provided at the second threaded portion 21, or the coil spring 24 may be provided between the snap ring and the driven member 7. The screw thread of the first threaded portion 20 may be coated with nylon resin manufactured by Nylock Co. in U.S.A., having a large elastically repulsion force and a frictional coupling force so as to restrict the forward screwing of driven member 7 by the locking effect of the resin coating. Alternatively, the driven member 7 may be fixed to the driving shaft 5 by screwing a bolt or driving a cotter pin. Thus, the spring 24 is not inevitably required.

Next, explanation will be given on operation of the lever type hoist and traction machine constructed as the above-mentioned.

Firstly, for hoisting a load, the operating part 15 provided at the operating lever 16 operates to engage the feed pawl of the pawl member 14 with the teeth 8a of the driving member 8 and the lever 16 is operated in swinging motion, thereby normally rotating the driving member 8. The driving member 8, when normally rotating, screws forward leftwardly in Fig.1, that is, toward the driven member 7, the mechanical brake 13 operates, and the driving torque of driving member 8 is transmitted from the driving shaft 5 to the load sheave 3 through the reduction gear mechanism 6 and tubular shaft 4, so that the hoisting work of a load connected to the chain engaging with the load sheave 3 is performed following the rotation thereof.

When the load is lowered, a reverse rotating pawl of the pawl member 14 at the operating part 15 is engaged with one of the teeth 8a of the driving member 8 to swing the lever 16, thereby reversely rotating the driving member 8. Since the engaging projections 31 are put in the positions X shown by the dotted lines in Fig.3, in other words, between the first regulating surface 33 and the second regulating surface 34, the driving member 8 relative-rotates with respect to the driving shaft 5 between the first regulating surface 33 and the second regulating surface 34 so as to be rearwardly movable with respect to the driven member 7. hence, the driven member 7 backwardly moves to stop braking action of mechanical brake 13 and the

driving shaft 5 can reversely rotate only to an extent of reverse rotation of driving member 8, thereby performing the load lowering work in safety.

In addition, during the load hoisting or lowering work, the operating handle 18 is rotated normally or reversely without being pulled toward the stopper 17 against the elastic biasing member 19, whereby the driving member 8 is moved in the direction of operating or not-operating the mechanical brake 13 with respect to the driving shaft 5. Hence, the load sheave 3 is rotated normally or reversely only by a rotation angle corresponding to rotation of operating handle 18, thereby enabling a pulling-out amount or a winding-up amount of the chain to be finely adjusted.

Next, explanation will be given on a case where the load sheave 3 is put in the free rotation state to freely extend or reduce a length of the chain toward the load.

At first, the reversing pawl of the pawl member 14 engages with one of the teeth 8a of driving member 8 and, when the operating handle 18 normally rotates, the driving member 8 is fixed not to rotate together with the operating handle 18. In this state, the operating handle 18 is pulled out toward the stopper 17 against the elastic biasing member 19, in other words, the same is moved from the first position as shown in Fig.1, to the second position apart from the driving member 8, thereby allowing normal rotating of the operating handle 18. At this time, while the driving member 8 cannot normally rotate because the reversing pawl of pawl member 14 engages with the tooth 8a at the driving member 18, the driving shaft 5 screwing with the driving member 8 rotates together with the operating handle 18 through the stopper 17. In other words, the driving shaft 5 normally relative-rotates with respect to the driving member 8 and in excess of the range regulated by the first and second regulating surfaces 33 and 34. Accordingly, the driving member 8 axially moves away from the driven member 7 in Fig.1 so that the braking action by the mechanical brake 13 can be released and the load sheave 3 can be put in the free rotation state, in which the chain, when pulled toward the load side, can quickly be extended at the load side and, when pulled toward the no-load side, can quickly be shortened at the load side.

As above-mentioned, the operating handle 18 is pulled out and rotatably operated so that the engaging projections 31 provided at the operating handle 18 can rotatably be moved to the position Y shown by the dotted line in Fig.3. In, this state, since the operating handle 18 is biased toward the driving member 8 by the elastic biasing member 19, so that the projecting utmost ends of engaging projections 31 elastically contact with the free rota-

tion control surfaces 35 of projecting portions 32 provided at the driving member 8 as shown in Fig.5, whereby frictional resistance caused by the elastic contact can hold the load sheave 3 in the free rotation state. Accordingly, when the chain is adjusted by holding such the free rotation state, the input range of a pulling force of the chain can be more extended than a conventional example, whereby the chain at the load side can be pulled or contracted without requiring skill. In addition, in the first embodiment, an elastic ring 37 is interposed between the outer peripheral surface of the first sleeve 25 and the driving member 8 so that the load sheave 3 can further be easy to hold the free rotation state thereof by the relative rotation resistance of the driving member 8 with respect to the first sleeve 25.

The regulating portions 36 are provided at the projecting portions 32 provided at the driving member 8 so that, when the driving member 8 relative-rotates with respect to the driving shaft 5 in the state where the utmost end faces of the engaging projections 31 are in elastic contact with the free rotation control surfaces 35 of the projecting portions 32, the front of each engaging projection 31 in the rotation direction thereof is regulated of its further rotation by the regulating portion 36, whereby, when the operating handle 18 is relative-rotated with respect to the driving member 8 for freely rotating the load sheave 3, the front of each engaging projection 31 in the rotation direction thereof engages with the regulating portion 36 so as to restrict its rotational angle and a space between the driving member 8 and the driven member 7 can be restricted not to be wider than required to freely rotate the load sheave 3. Accordingly, when the load sheave 3 freely rotates through the operating handle 18 rotating relatively with respect to the driving member 8, the free rotation operation is performable without uselessly rotating the operating handle 18 more than required. Also, when the chain is excessively pulled toward the load and the stopper provided at the no-load end of chain engages with the side plate 1 or 2 to incapacitate further pulling of chain so as to abruptly stop the rotation of driving shaft 5, the driving member 8 rotates by its inertia force and further screws rightwardly. As the result, the utmost end faces of engaging projections 31 elastically contact further strongly with the free rotation control surfaces 35 at the projecting portions 32 in a biting manner, thereby avoiding incapacitation of release of free rotation control.

Furthermore, in the state of the free rotation control as the above-mentioned, when the pulling force of chain is strengthened to apply a strong force in the reverse direction onto the load sheave 3, the elastic contact of the projecting utmost end

face of each engaging projection 31 with each free rotation control surface 35 is released so that each engaging projection 31 returns to between the first regulating surface 33 and the second regulating surface 34 and, as the above-mentioned, returns to the state where the mechanical brake 13 exerts or stops the braking action. In other words, during the free rotation, when the load sheave 3 is subjected to a strong force in the reverse direction, the driving member 8 screws with the driving shaft 5 and its rotary inertia force is larger than that of the driving shaft 5, whereby the free rotation control surfaces 35 slide with respect to the engaging projections 31 and the driving member 8 starts rotation somewhat later than the rotation of operating handle 18. As the result, the elastic contact of the respective projecting utmost end faces of engaging projections 31 with the free rotation control surfaces 35 is released, resulting in that each engaging projection 31 returns to between the first regulating surface 33 and the second regulating surface 34. In addition, in this case, the operating handle 18 overcomes the relative rotational resistance of the projecting utmost end faces of the engaging projections 31 with respect to the free rotation control surfaces 35 and the relative rotational resistance by the elastic ring 37, thereby rotating relatively in the reverse rotation direction with respect to the driving resistance 8. Hence, an input range of the pulling force for the chain during the free rotation control is widened so that the free rotation control is performable without requiring skill.

When the chain engaging with the load sheave 3 is subjected to a load and the load sheave 3 is applied with the load in the reverse rotation, even though the operating handle 18 is operated to carry out free rotation, the operating handle 18 together with the driving shaft 5 relative-rotates in the reverse rotation direction by the above-mentioned load, so that the elastic contact of the utmost end faces of the engaging projection 31 with the free rotation control surfaces 35 is released, thereby returning to the state where the mechanical brake 13 exerts or stops the braking action. Accordingly, the load sheave 3 cannot be put in the free rotation state, thereby improving safety.

Next, explanation will be given on a second embodiment of the invention shown in Figs.6 through 9.

The second embodiment assembles an overload prevention mechanism in the first embodiment, and is similar in the fundamental construction to the first embodiment. Accordingly, the constitution in common with the first embodiment is omitted from its description and the common components are designated with the same reference numerals.

In the second embodiment, the driving member 8 in the first embodiment comprises a first driving member 41 having a boss 41a screwable with a driving shaft 5 and a larger diameter portion 41b opposite to a braking plate 9 of the mechanical brake 13 and a second driving member 42 relative-rotatably supported on the outer periphery of the boss 41a, and at the outer periphery of the second driving member 42 are provided teeth 42a engageable with a pawl member 14 provided at the operating lever 16.

At the boss 41a of the first driving member 41 are disposed a pair of friction plates 43 and 44 in a manner of longitudinally sandwiching the second driving member 42 therebetween, an elastic member 46 of a disc spring is disposed outside one friction plate 44 through a holding plate 45, and an adjusting member 47 for changing a biasing force of the elastic member 46 to the friction plates 43 and 46 and for adjusting a slip load screws with the boss 41a outside the elastic member 46, thereby constituting the overload prevention mechanism 40.

In detail, the first driving member 41 is provided at one axial end of the boss 41a with the larger diameter portion 41b having a biasing surface opposite to the braking plate 9 and at the other axial end of boss 41a with a smaller diameter portion 41c having a screw thread at the outer periphery, and the elastic member 46 is free-fitted onto the smaller diameter portion 41c and the adjusting member 47 screws therewith. A locking groove 41d for the holding plate 45 is provided at the outer periphery of the boss 41a and a projection projecting from the inner periphery of the holding plate 45 is fitted into the groove 41d, the holding plate 45 being supported to the boss 41a in relation of being axially movable and not relative-rotatable.

The second driving member 42 comprises a cylindrical member 42c having a vertical portion 42b and the teeth 42a, the vertical portion 42b is supported at the inner periphery thereof rotatably onto the boss 41a, and between the inner periphery of the vertical portion 42b and the outer periphery of the boss 41a is provided a unidirectional rotation mechanism which, when the second driving member 42 rotates in the driving direction, makes the second driving member 42 freely rotatable with respect to the first driving member 41 and, when rotating in the not-driving direction, makes the same rotatable integrally with the first driving member 41.

The unidirectional rotation mechanism, as shown in Fig.9, is so constructed that a recess 48 is formed at the outer periphery of the boss 41a at the first driving member 41, an engaging member 49 is held in the recess 48 in relation of being biased always radially outwardly of the boss 41a

through a spring 50, and at the inner periphery of the second driving member 42 are formed a plurality (eight in Fig.9) of engaging grooves which each allow the engaging member 49 to enter therein and extends circumferentially in a wedge-like manner, so that when the second driving member 42 is rotated in the chain lowering direction as shown by the arrow in Fig.9, the engaging member 49 engages with one of the engaging grooves 51 at an angle of at least 45° or more and the second driving member 42 and the first driving member 41 are combined with each other to be integrally rotatable, thereby coping with the case where a torque larger than a transmitting torque of the overload prevention mechanism 40 during the lowering the chain is required.

The second embodiment of the invention constructed as the above-mentioned assemblies therein the overload prevention mechanism 40 and also a free rotation control apparatus as the same as the first embodiment, the free rotation control apparatus is not different from that in the first embodiment, thereby omitting description thereof. The adjusting member 47 of the overload prevention mechanism 40 is disposed opposite to the operating handle 18 at the free rotation control apparatus.

At the adjusting member 47 are provided regulation portions 52 for regulating a relative rotation range of the first driving member 41 with respect to the driving shaft 5 in the first position of the operating handle 18, and free rotation control surfaces 53 which come in elastic contact with the engaging projections 31 provided at the operating handle 18, apply resistance to the rotation of the first driving member 41 with respect to the driving shaft 5, and hold the free rotation of the driving shaft 5 by the operating handle 18, so that the adjusting member 47 may adjust a slip load and also hold the free rotation control at the overload prevention mechanism 40. In greater detail of the adjusting member 47, the regulation portions 52, as shown in Figs.6 and 8, are symmetrically cut out at the outer periphery and regulating surfaces 54 and 55 are formed at both circumferential sides of each cutout, so that when the operating handle 18 is not operated, in other words, in the first position, each engaging projection 31 of the operating handle 18, as the same as the first embodiment, enters into the cutout to engage with the regulating surface 54 or 55, thereby regulating the relative rotation range of the first driving member 41 with respect to the driving shaft 5. Accordingly, within the relative rotation range, the first driving member 41 can screw forward or backward with respect to the braking plate 9 and the mechanical brake 13 operates to allow the driving shaft 5 to rotate following the rotations of the first and second driving members 41 and 42, thereby enabling the load to be hoisted,

lowered, hauled, or traction-released.

On the opposite surface of the adjusting member 47 to the operating handle 18 and at the front in the normal rotation direction with respect to the regulation portions 52 as shown by the arrow in fig.8 are symmetrically provided the free rotation control surfaces 53 in elastic contact with the utmost end faces of engaging projections 31 in the second position of the operating handle 18 respectively, the elastic contact of the engaging projections 31 with the free rotation control surfaces 53 applies resistance to the rotation of the first driving member 41 through the adjusting member 47, thereby enabling the free rotation operation by the operating handle 18 to be held.

Also, in this case, the second driving member 42, as in the first embodiment, is fixed through the pawl member 14 at the lever 16, and then the operating handle 18 is drawn out toward the stopper 17, and relative-rotated with respect to the first and second driving members 41 and 42, whereby the driving shaft 5 rotates integrally therewith. Hence, the first driving member 41 screwable with the driving shaft 5 screws backwardly from the braking plate 9, whereby the driving shaft 5 can be put in the free rotation state, at which time the utmost end faces of projections 31, as shown in Figs.7 and 8, come into elastic contact with the free rotation control surfaces 53 respectively, whereby the first driving member 41 can be restrained from the relative rotation thereof with respect to the driving shaft 5 and the free rotation state of the driving shaft 5 can be held by the restraint.

In addition, in the second embodiment, as shown in Fig.8, free rotation regulating portions 56 are provided which, when the operating handle 18 is rotated with respect to the first and second driving members 41 and 42, prevent the operating handle 18 from rotating by contact of each projection 31 more than required.

Other than the construction of assembling an overload prevention mechanism 40, the second embodiment is different from the first embodiment in the following points: Firstly, the stopper 17 integrally forms at its center a cylindrical member 17a serration-coupled with serrations 20 at the driving shaft 5, and the sleeve 25 in the first embodiment is omitted.

Also, the flange 25a at the sleeve 25 of the first embodiment is not provided at the cylindrical member 17a, whereby the operating handle 18 is biased by the elastic biasing member 19 so as to bring the handle 18 into elastic contact with the end face of a smaller diameter portion 41c at the first driving member 41.

Furthermore, a driven member 7 screws with the driving shaft 5 and is restrained by use of a

snap ring 57 from its axial movement.

Next, explanation will be given on operation of the second embodiment constructed as the above-mentioned.

At first, for the hoisting or traction of load, the feed pawl at the pawl member 14 provided at the operating lever 16 engages with tooth 42a at the second driving member 42 by operating the control portion 15 so as to swing the lever 16, whereby the second driving member 42 is rotated and the first driving member 41 together therewith is normally rotated through the overload prevention mechanism 40. In this case, since the projections 31, as shown by the dotted lines in Fig.8, are positioned at the regulation portions 52 and between the regulating surfaces 54 and 55, the first driving member 41, when normally rotating, screws toward the braking plate 9 and the mechanical brake 13 operates. A driving torque of the second driving member 42 is transmitted to the first driving member 41 through the overload prevention mechanism 40, and to the driving shaft 5 through the mechanical brake 13, and also transmitted from the driving shaft 5 to the load sheave 3 through the reduction gear mechanism 6 and tubular shaft 4, thereby enabling the hoisting or traction of load. In this state, when the load sheave 3 is subjected to a load larger than the rating load adjusted by the adjusting member 47, the overload prevention mechanism 40 slips to eliminate power transmission to the first driving member 41, thereby enabling the hoisting or the traction over the rating to be regulated.

In a case where the chain lowering or the release of traction is performed, the reverse rotation pawl at the pawl member 14 engages with one of the teeth 42a of the second driving member 42 so as to swing the lever 16, whereby the first driving member 41 is reversely and integrally rotated with the second driving member 42 through a unidirectional rotation mechanism. In this case, since the projections 31 are positioned at the regulation portions 52, the first driving member 41 relative-rotates with respect to the driving shaft 5 to be backwardly screwable with respect to the braking plate 9, so that the driving shaft 5 can be rotated at a predetermined angle until the mechanical brake 13 operates, thereby enabling the hoisting or traction of the chain.

In this case, the lever 16 is operated in swinging motion to rotate the first and second driving members 41 and 42 in the not-driving direction, that is, to reversely rotate them, thereby reversely rotating the first driving member 41. As shown in Fig.9, at the inner periphery of the second driving member 42, a plurality of the engaging grooves 51 engageable with the engaging member 9 are provided spaced at equal intervals, so that the engaging member 49 engages with one engaging groove

51 at an angle of at least 45° and without the need of once rotating the second driving member 42, thereby enabling the second driving member 42 to be integral with the first driving member 41 and to quickly start the lowering of chain or the release of traction to that extent.

Next, explanation will be given on a case where the driving shaft 5 is put in the free rotation state so as to carry out free extension or contraction of the chain at the load side.

Such operation, as the same as the first embodiment, is carried out in such a manner that the reverse rotation pawl of the pawl member 14 engages with the teeth 42a of the second driving member 42 and, when the operating handle 18 normally rotates, the second driving member 42 is made not to rotate together with the operating handle 18, and then the operating handle 18 is pulled toward the stopper 17 against the elastic biasing member 19 and normally rotated. At this time, although the second driving member 42, whose tooth 42a engages with the reverse rotation pawl of the pawl member 14, cannot normally rotate, the driving shaft 5 together with the operating handle 18 is relative-rotated in the normal direction through the stopper 17 in excess of the ranges regulated by the regulation portions 52. The first driving member 41 is moved, by the relative rotation, away from the braking plate 9, that is, rightwardly in Fig. 6, whereby the braking action by the mechanical brake 13 can be released to put the driving shaft 5 in the free rotation state. The elastic biasing member 19 biases the projecting utmost end faces of the projections 31 to come into elastic contact with the free rotation control surfaces 53 at the adjusting member 47 as shown in Figs. 7 and 8, whereby the operating handle 18 can be restricted from its relative rotation with respect to the first and second driving members 41 and 42. Hence, it is possible to keep the driving shaft 5 in the free rotation state. Accordingly, the chain, when pulled to the load side in this state, can quickly be extended and, when pulled to the no-load side, can quickly be contracted.

In addition, during the free rotation of driving shaft 5, the projections 31 come into elastic contact with the free rotation control surfaces 53, but the adjusting member 47 does not rotate to axially move by this elastic contact and does not change the rating load on which the overload prevention mechanism 40 starts its operation. In other words, since the adjusting member 47 is subjected to the reaction force of the elastic member 46, the rotational resistance of adjusting member 47 is larger than that when the operating handle 18 is in elastic contact at the projections 31 thereof with the free rotation control surfaces 53 relative-rotates with respect to the first driving member 41, whereby the

adjusting member 47 never rotates by a torque transmitted thereto through the projections 31. Accordingly, a slip load of the overload prevention mechanism 40 preadjusted by the adjusting member 47 never changes.

Furthermore, in the state where the driving shaft 5 is held in the free rotation state as the above-mentioned, when a pulling force of the chain is strengthened to apply to the driving shaft 5 a strong force in the reverse rotation direction, the elastic contact of the respective utmost end faces of the projections 31 with respect to the respective free rotation control surfaces 53 is released, whereby the projections 31 return to the regulating portions 52 so as to return to the state where the mechanical brake 13 can operate.

As the above-mentioned, the second embodiment, when the operating handle 18 is operated in not-free-rotation, allows the first driving member 41 to screw forward and backward with respect to the braking plate 9 so as to operate the mechanical brake 13, whereby the hoisting, lowering, traction of the load, and release of traction are performable and also the overload prevention mechanism 40 is simultaneously performable. Moreover, the operating handle 18, when freely rotating, is relative-rotated with respect to the first and second driving members 41 and 42 as the same as the first embodiment and the projections 31 at the operating handle 18 are brought into elastic contact with the free rotation control surfaces 53 to enable the free rotation of the driving shaft 5 to be held.

Accordingly, the overload prevention mechanism 40 can operate to perform the overload prevention and also the free rotation operating handle 18 can hold the driving shaft 5 in the free rotation state without requiring skill. Moreover, the adjusting member 47 is used not only as part for adjusting the rating load of the overload prevention mechanism 40 but also as part for holding the driving shaft 5 in the free rotation state, thereby saving the number of parts to that extent.

In addition, in the above-mentioned second embodiment, as shown in Fig. 9, the engaging member 49 is held in the recess 48 at the outer periphery of the boss 41a of the first driving member 41 and the engaging grooves 51 are provided at the inner periphery of the second driving member 42, but the engaging member 49 may be held at the second driving member 42 and a plurality of engaging grooves may be provided at the outer periphery of the boss 41a.

As seen from the above, the hoist and traction machine of the present invention can release the braking action of the mechanical brake and perform the free rotation control by the free rotation operation in which the operating handle 18 is moved away from the driving member 8 against the elastic

biasing member 19 and normally rotated, and also can hold the state of releasing the braking action of the mechanical brake, in brief, the free rotation control by being biased by the elastic biasing member 19. Accordingly, the input range of pulling force of chain during the free rotation control is expanded to ensure the free rotation control without requiring skill. Moreover, since the operating handle 18 is adapted to operate to put the load sheave 3 in the free rotation state, the operating handle 18 can enlarge a ratio of radius of movement of its rotation operation and perform the free rotation with a light force in comparison with the direct rotation of the driving shaft 5.

Accordingly, free extension or contraction of the chain with respect to the load side can simply be carried out without requiring skill.

When the chain engaged with the load sheave 3 is subjected to the load, even though the free rotation operation is intended to be performed, the free rotation state cannot be held, thereby increasing safety. Also, as described in the second embodiment, the hoist and traction machine assembling therein the overload prevention mechanism 40 can perform the overload prevention by operating the overload prevention mechanism 40 and also can operate the operating handle 18 to hold the driving shaft 5 in the free rotation state without requiring skill. Moreover, the adjusting member 47 is used not only as part for adjusting the rating load of the overload prevention mechanism 40 but also as part for holding the driving shaft 5 in the free rotation state, thereby saving the number of parts to that extent.

Also, between the first and second driving members 41 and 42 is provided the unidirectional rotation mechanism which, when the second driving member 42 rotates in the driving direction, makes the second driving member 42 freely rotatable with respect to the first driving member 41 and, when rotating in the not-driving direction, makes the second driving member 42 integrally rotatable with the first driving member 41. A plurality of engaging grooves 51 engageable with the engaging member 49 constituting the unidirectional rotation mechanism are provided, whereby, when the second driving member 42 is reversely rotated to reversely rotate the first driving member 41 to thereby carry out the lowering of chain or release of traction, the engaging member 49 is engaged with one of the engaging grooves 51 through a small angle without requiring a full rotation of the second driving member 42 and can be integral with the first driving member 41, whereby the lowering of chain or release of traction can quickly be started to that extent.

Although the invention has been described with reference to several different embodiments, these

embodiments are merely exemplary and not limiting of the invention which is defined solely by the appended claims.

Claims

1. A hoist and traction machine comprising a load sheave (3); a driving shaft (5), having a driven member (7), for driving said load sheave (3); a driving member (8,41;42) threadingly mounted to said driving shaft (5); a braking pawl (12); a braking ratchet wheel (11) engageable with said braking pawl (12); braking plates (9,10) interposed between said driving member (8,41;42) and said driven member (7), constituting a mechanical brake; driving means for normally and reversely driving said driving member; a cover (13a) for covering said mechanical brake, wherein said driving member does not contact said cover, said hoist and traction machine further comprising a free rotation control device for controlling operation of said mechanical brake so as to provide free rotation for said load sheave (3), said free rotation control device comprising:

(a) a stopper (17) provided at an axial end of said driving shaft (5) so as to be non-rotatable and axially immovable with respect to said driving shaft (5);

(b) a free-rotation operation handle (18) provided between said stopper (17) and said driving member (8,41;42) in such a manner as to be axially movable between a first position, in proximity to said driving member, and a second position, apart therefrom;

(c) regulation means, for regulating a relative rotation range of said driving member with respect to said driving shaft (5) when said handle (18) is put in the first position, characterized in that

(i) an elastically biasing member (19) is interposed between said stopper (17) and said handle (18), for biasing said handle (18) toward said first position in proximity to said driving member,

(ii) said regulation means is provided between said handle (18) and said driving member (8,41;42), so that, when said handle is put in the second position, the regulation is released;

(iii) free-rotation-control holding means (31,35;53) is provided between said handle (18) and the driving member (8,41;42), for applying a biasing force to said driving member from said elastically biasing member (19), when said handle (18) is put in the second position to release said regulation and is rotated so as



biasing member 19 and normally rotated, and also can hold the state of releasing the braking action of the mechanical brake, in brief, the free rotation control by being biased by the elastic biasing member 19. Accordingly, the input range of pulling force of chain during the free rotation control is expanded to ensure the free rotation control without requiring skill. Moreover, since the operating handle 18 is adapted to operate to put the load sheave 3 in the free rotation state, the operating handle 18 can enlarge a ratio of radius of movement of its rotation operation and perform the free rotation with a light force in comparison with the direct rotation of the driving shaft 5.

Accordingly, free extension or contraction of the chain with respect to the load side can simply be carried out without requiring skill.

When the chain engaged with the load sheave 3 is subjected to the load, even though the free rotation operation is intended to be performed, the free rotation state cannot be held, thereby increasing safety. Also, as described in the second embodiment, the hoist and traction machine assembling therein the overload prevention mechanism 40 can perform the overload prevention by operating the overload prevention mechanism 40 and also can operate the operating handle 18 to hold the driving shaft 5 in the free rotation state without requiring skill. Moreover, the adjusting member 47 is used not only as part for adjusting the rating load of the overload prevention mechanism 40 but also as part for holding the driving shaft 5 in the free rotation state, thereby saving the number of parts to that extent.

Also, between the first and second driving members 41 and 42 is provided the unidirectional rotation mechanism which, when the second driving member 42 rotates in the driving direction, makes the second driving member 42 freely rotatable with respect to the first driving member 41 and, when rotating in the not-driving direction, makes the second driving member 42 integrally rotatable with the first driving member 41. A plurality of engaging grooves 51 engageable with the engaging member 49 constituting the unidirectional rotation mechanism are provided, whereby, when the second driving member 42 is reversely rotated to reversely rotate the first driving member 41 to thereby carry out the lowering of chain or release of traction, the engaging member 49 is engaged with one of the engaging grooves 51 through a small angle without requiring a full rotation of the second driving member 42 and can be integral with the first driving member 41, whereby the lowering of chain or release of traction can quickly be started to that extent.

Although the invention has been described with reference to several different embodiments, these

embodiments are merely exemplary and not limiting of the invention which is defined solely by the appended claims.

5 Claims

1. A hoist and traction machine comprising a load sheave (3); a driving shaft (5), having a driven member (7), for driving said load sheave (3); a driving member (8,41;42) threadingly mounted to said driving shaft (5); a braking pawl (12); a braking ratchet wheel (11) engageable with said braking pawl (12); braking plates (9,10) interposed between said driving member (8,41;42) and said driven member (7), constituting a mechanical brake; driving means for normally and reversely driving said driving member; a cover (13a) for covering said mechanical brake, wherein said driving member does not contact said cover, said hoist and traction machine further comprising a free rotation control device for controlling operation of said mechanical brake so as to provide free rotation for said load sheave (3), said free rotation control device comprising:

(a) a stopper (17) provided at an axial end of said driving shaft (5) so as to be non-rotatable and axially immovable with respect to said driving shaft (5);

(b) a free-rotation operation handle (18) provided between said stopper (17) and said driving member (8,41;42) in such a manner as to be axially movable between a first position, in proximity to said driving member, and a second position, apart therefrom;

(c) regulation means, for regulating a relative rotation range of said driving member with respect to said driving shaft (5) when said handle (18) is put in the first position, characterized in that

(i) an elastically biasing member (19) is interposed between said stopper (17) and said handle (18), for biasing said handle (18) toward said first position in proximity to said driving member,

(ii) said regulation means is provided between said handle (18) and said driving member (8,41;42), so that, when said handle is put in the second position, the regulation is released;

(iii) free-rotation-control holding means (31,35;53) is provided between said handle (18) and the driving member (8,41;42), for applying a biasing force to said driving member from said elastically biasing member (19), when said handle (18) is put in the second position to release said regulation and is rotated so as



to perform the free rotation operation, so as to hold said free rotation operation; and

(iv) said handle (18) is non-rotatable relative to said driving shaft (5) due to engagement with said stopper (17), said handle (18) being rotationally operable to rotate said driving shaft (5) to cause said driving member (8,41;42) to screw backwards in the non-braking direction, so as to provide the free rotation operation.

2. A hoist and traction machine according to claim 1, wherein said regulation means is provided with a pair of projecting portions having regulating surfaces (33,34) for regulating a relative rotation range of said driving member (8,41;42) with respect to said driving shaft (5) and engaging projections (31) fitted between said projecting portions and engageable with said regulating surfaces respectively, and said free rotation control holding means is provided with free rotation regulating surfaces (53) positioned out of a regulation range by each regulating surface, so that when said operating handle (18) is put in the second position to operate free rotation, said engaging projections (31) are adapted to come into elastic contact with said control surfaces (53) respectively.
3. A hoist and traction machine according to claim 2 wherein said free rotation control holding means (31,35;53) is provided on said free rotation control surfaces and at the front in the free rotation operating direction of said operating handle with regulation portions for regulating the free rotation operation range by said operating handle (18).
4. A hoist and traction machine according to claim 1, wherein said driving shaft (5) has a positioning portion for setting the first position of said operating handle (18), the first position by said positioning portion being set in a position where said operating handle is in no contact with said driving member (8,41;42).
5. A hoist and traction machine according to claim 1, further provided with an overload prevention mechanism (40) comprising:

said driving member comprising a first driving member (41) having a boss screwably with said driving shaft (5) and a larger diameter portion opposite to said braking plate at said mechanical brake and a second driving member (42) relative-rotatably supported to said boss of said first driving member, said boss (41a) at said first driving member supporting

friction plates and an elastic member and screwing with an adjusting member (47) for changing a biasing force of said friction plates caused by said elastic member so as to adjust a slip load, said adjusting member (47) provided with regulation portions for regulating a relative rotation range of said driving member with respect to said driving shaft at the first position of said operating handle and free rotation control surfaces with which said handle comes into elastic contact and for holding the free rotation of said driving shaft by said operating handle.

6. A hoist and traction machine according to claim 5, wherein said regulation portions (52) of said adjusting member (45) each comprise a cutout having a pair of regulating surfaces (54,55) for regulating the relative rotation range of said driving member with respect to said driving shaft, said operating handle (18) being provided with engaging projections (31) for entering into said cutouts to engage with said regulating surfaces at the first position of said operating handle and for coming in elastic contact with said free rotation control surfaces at the second position of said operating handle.
7. A hoist and traction machine according to claim 6, wherein said adjusting member (47) is provided at the front of each of said free rotation control surfaces and at the front in the free rotation operation direction of said operating handle with regulation portions for regulating a free rotation operation range by said operating handle.
8. A hoist and traction machine according to claim 5, wherein between said boss (41a) of said first driving member and said second driving member is provided a unidirectional rotation mechanism (49) which, when said second driving member rotates in the driving rotation direction, makes said second driving member freely rotatable with respect to said first driving member and, when the same rotates in the not-driving rotation direction, integrally rotates said second driving member with said first driving member, said unidirectional rotation mechanism comprising an engaging member (49) held by one of said first and second members in relation of being projectable therefrom or retractable therein and an engaging groove with which said engaging member engages when said second driving member rotates in the not-driving rotation direction, said engaging groove being provided in a plurality and circumferentially.



Patentansprüche

1. Hebe- und Zugmaschine mit einer Lastseilscheibe (3), einer Antriebswelle (5) mit einem angetriebenen Element (7) zum Antrieb der Lastseilscheibe (3); einem durch Verschrauben an der Antriebswelle (5) angebrachten Antriebselement (8, 41; 42); einer Bremsklinke (12); einem mit der Bremsklinke (12) in Eingriff bringbaren Bremssperrad (11); zwischen dem Antriebselement (8, 41; 42) und dem angetriebenen Element (7) angeordneten Bremsscheiben (9, 10), welche eine mechanische Bremse bilden; Antriebsmitteln zum Normal- und Rückwärtsantrieb des Antriebselements; einer Abdeckung (13a) zum Abdecken der mechanischen Bremse, wobei das Antriebselement nicht mit der Abdeckung in Berührung steht, wobei die Hebe- und Zugmaschine des weiteren eine Vorrichtung zur Steuerung der freien Rotation umfaßt, um die Funktion der mechanischen Bremse zu steuern und somit der Lastseilscheibe (3) eine freie Rotation zu ermöglichen, wobei die Vorrichtung zur Steuerung der freien Rotation umfaßt:

(a) einen Anschlag (17), welcher an einem axialen Ende der Antriebswelle (5) zur Verfügung gestellt ist, um nicht drehbar und axial unbeweglich in bezug auf die Antriebswelle (5) zu sein;

(b) einen Bedienungshebel (18) für freie Rotation, welcher zwischen dem Anschlag (17) und dem Antriebselement (8, 41; 42) so vorgesehen ist, um zwischen einer ersten Position in der Nähe des Antriebselements und einer zweiten, sich davon entfernt befindlichen Position, axial bewegbar zu sein;

(c) Reguliermittel zur Regulierung eines Bereichs relativer Rotation des Antriebselements in bezug auf die Antriebswelle (5), wenn der Hebel (18) in die erste Position gebracht ist, dadurch gekennzeichnet daß

(i) ein elastisch spannendes Element (19) zwischen dem Anschlag (17) und dem Hebel (18) angeordnet ist, um den Hebel (18) zur ersten Position in der Nähe des Antriebselements hin zu spannen,

(ii) das Reguliermittel zwischen dem Hebel (18) und dem Antriebselement (8, 41; 42) zur Verfügung gestellt ist, so daß, wenn der Hebel in die zweite Position gebracht ist, die Regulierung gelöst ist;

(iii) zwischen dem Hebel (18) und dem Antriebselement (8, 41; 42) ein Haltemittel (31, 35; 53) für die Steuerung der freien Rotation zur Verfügung gestellt ist, um auf das Antriebselement eine Spannkraft von dem elastisch spannenden Ele-

ment (19) auszuüben, wenn der Hebel (18) in die zweite Position gebracht ist, um die Regulierung zu lösen, und gedreht wird, um die Funktion der freien Rotation auszuführen, um somit die Funktion der freien Rotation zu halten; und

(iv) der Hebel (18) auf Grund des Eingreifens mit dem Anschlag (17) relativ zur Antriebswelle (5) nicht drehbar ist, wobei der Hebel (18) drehbar bedienbar ist, um die Antriebswelle (5) zu drehen, um ein Rückwärtsschrauben des Antriebselements (8, 41; 42) in die Nicht-Bremsrichtung zu bewirken, um die Funktion der freien Rotation zu ermöglichen.

2. Hebe- und Zugmaschine gemäß Anspruch 1, bei der das Reguliermittel mit einem Paar vorstehender Bereiche mit Regulieroberflächen (33, 34) zur Regulierung eines Bereichs relativer Rotation des Antriebselements (8, 41; 42) in bezug auf die Antriebswelle (5) und mit eingreifenden Vorsprüngen (31), welche sich zwischen den vorstehenden Bereichen befinden und jeweils mit den Regulieroberflächen in Eingriff bringbar sind, versehen ist, und wobei das Haltemittel für die Steuerung der freien Rotation mit Oberflächen (53) zur Regulierung der freien Rotation versehen ist, welche durch jede Regulieroberfläche aus einem Regulierbereich heraus positioniert sind, so daß, wenn der Bedienungshebel (18) in die zweite Position für die Funktion der freien Rotation gebracht ist, die eingreifenden Vorsprünge (31) so ausgerichtet sind, um mit den Steuerflächen (53) jeweils in elastischen Kontakt zu kommen.

3. Hebe- und Zugmaschine gemäß Anspruch 2, bei der das Haltemittel (31, 35; 53) für die Steuerung der freien Rotation auf den Oberflächen zur Steuerung der freien Rotation sowie vorne in Richtung der Funktion der freien Rotation des Bedienungshebels mit Regulierbereichen zur Regulierung des Bereichs der Funktion der freien Rotation durch den Bedienungshebel (18) vorgesehen ist.

4. Hebe- und Zugmaschine gemäß Anspruch 1, bei der die Antriebswelle (5) einen Positionierbereich zur Einstellung der ersten Position des Bedienungshebels (18) aufweist, wobei die erste Position durch den Positionierbereich in einer Position eingestellt ist, in welcher der Bedienungshebel nicht mit dem Antriebselement (8, 41; 42) in Berührung steht.



5. Hebe- und Zugmaschine gemäß Anspruch 1, die des weiteren mit einem Mechanismus (40) zur Verhinderung von Überlastung versehen ist, welcher umfaßt:

das Antriebselement mit einem ersten Antriebselement (41) mit einem Vorsprung, welcher mit der Antriebswelle (5) verschraubbar ist, und einem Bereich von größerem Durchmesser gegenüber von der Bremsscheibe an der mechanischen Bremse und einem zweiten Antriebselement (42), welches an dem Vorsprung des ersten Antriebselements relativ drehbar gehalten ist, wobei der Vorsprung (41a) an dem ersten Antriebselement Reibungsscheiben und ein elastisches Element trägt und mit einem Einstellelement (47) zur Veränderung einer durch das elastische Element verursachten Spannkraft der Reibungsscheiben, um eine Schlupflast einzustellen, verschraubt, wobei das Einstellelement (47) mit Regulierbereichen zur Regulierung eines Bereichs relativer Rotation des Antriebselements in bezug auf die Antriebswelle an der ersten Position des Bedienungshebels und Oberflächen zur Steuerung der freien Rotation, mit welchen der Hebel in elastischen Kontakt kommt, und die dazu dienen, die freie Rotation der Antriebswelle durch den Bedienungshebel zu halten, versehen ist.

6. Hebe- und Zugmaschine gemäß Anspruch 5, bei der die Regulierbereiche (52) des Einstellelements (45) jeweils einen Ausschnitt mit einem Paar Regulieroberflächen (54, 55) zur Regulierung des Bereichs relativer Rotation des Antriebselements in bezug auf die Antriebswelle aufweisen, wobei der Bedienungshebel (18) mit eingreifenden Vorsprüngen (31) zum Eintreten in die Ausschnitte versehen ist, um mit den Regulieroberflächen an der ersten Position des Bedienungshebels in Eingriff zu stehen und mit den Oberflächen zur Steuerung der freien Rotation an der zweiten Position des Bedienungshebels in elastischen Kontakt zu kommen.

7. Hebe- und Zugmaschine gemäß Anspruch 6, bei der das Einstellelement (47) an der Vorderseite jeder der Oberflächen zur Steuerung der freien Rotation sowie vorne in Richtung der Funktion der freien Rotation des Bedienungshebels mit Regulierbereichen zur Regulierung eines Bereichs der Funktion der freien Rotation durch den Bedienungshebel vorgesehen ist.

8. Hebe- und Zugmaschine gemäß Anspruch 5, bei der zwischen dem Vorsprung (41a) des ersten Antriebselements und dem zweiten An-

triebselement ein Einrichtungsrotationsmechanismus (49) zur Verfügung gestellt ist, welcher, wenn das zweite Antriebselement sich in die Antriebsrotationsrichtung dreht, das zweite Antriebselement in bezug auf das erste Antriebselement frei drehbar macht und, wenn dasselbe sich in die Nicht-Antriebsrichtung dreht, einstückig das zweite Antriebselement mit dem ersten Antriebselement dreht, wobei der Einrichtungsrotationsmechanismus ein Eingriffselement (49) aufweist, welches von einem der Elemente ausgewählt aus dem ersten und dem zweiten Element gehalten ist, wobei es davon überstehen kann oder darin einschiebbar ist, und eine Eingriffsnut, mit welcher das Eingriffselement in Eingriff steht, wenn das zweite Antriebselement sich in der Nicht-Antriebsrotationsrichtung dreht, wobei die Eingriffsnut in einer Mehrzahl und um den Umfang zur Verfügung gestellt ist.

Revendications

1. Engin de levage comprenant une poulie de charge (3) ; un arbre de commande (5), ayant un élément commandé (7), pour commander ladite poulie de charge (3) ; un élément de commande (8, 41 ; 42) monté de façon filetée sur ledit arbre de commande (5) ; un cliquet de freinage (12) ; une roue à rochet de freinage (11) en prise avec ledit cliquet de freinage (12) ; des plaquettes de freinage (9, 10) interposées entre ledit élément de commande (8, 41 ; 42) et ledit élément commandé (7), constituant un frein mécanique ; un dispositif de commande pour commander normalement et inversement ledit élément de commande ; un couvercle (13a) pour couvrir ledit frein mécanique, de sorte que ledit élément de commande n'est pas en contact avec ledit couvercle, ledit engin de levage comprenant en outre un dispositif de commande de rotation libre pour commander le fonctionnement dudit frein mécanique afin de fournir une rotation libre pour ladite poulie de charge (3), ledit dispositif de commande de rotation libre comprenant :
 - (a) un taquet (17) placé à une extrémité axiale dudit arbre de commande (5) afin d'être non-tournant et axialement fixe par rapport audit arbre de commande (5) ;
 - (b) une poignée de manoeuvre de rotation libre (18) placée entre ledit taquet (17) et ledit élément de commande (8, 41 ; 42) afin d'être mobile axialement entre une première position, à proximité dudit élément de commande, et une seconde position, espacée de celle-ci ;



(c) un dispositif de régulation, pour réguler un intervalle de rotation relatif dudit élément de commande par rapport audit arbre de commande (5) lorsque ladite poignée (18) est mise dans la première position, caractérisé en ce que

(i) un élément de sollicitation élastique (19) est interposé entre ledit taquet (17) et ladite poignée (18) pour solliciter ladite poignée (18) en direction de ladite première position à proximité dudit élément de commande,

(ii) ledit dispositif de régulation est placé entre ladite poignée (18) et ledit élément de commande (8, 41 ; 42), de sorte que, lorsque ladite poignée est mise dans la seconde position, la régulation est libérée ;

(iii) un dispositif de maintien de commande de rotation libre (31, 35 ; 53) est placé entre ladite poignée (18) et l'élément de commande (8, 41 ; 42), pour appliquer une force de sollicitation audit élément de commande à partir dudit élément de sollicitation élastique (19), lorsque ladite poignée (18) est mise dans la seconde position pour libérer ladite régulation et est tournée afin de réaliser l'opération de rotation libre, afin de maintenir ladite opération de rotation libre ; et

(iv) ladite poignée (18) est non-tournante par rapport audit arbre de commande (5) du fait de l'engagement avec ledit taquet (17), ladite poignée (18) étant actionnable de façon tournante pour faire tourner ledit arbre de commande (5) pour provoquer le vissage vers l'arrière dudit élément de commande (8, 41 ; 42) dans la direction de non-freinage afin de fournir l'opération de rotation libre.

2. Engin de levage selon la revendication 1, dans lequel ledit dispositif de régulation est pourvu d'une paire de parties en saillie ayant des surfaces de régulation (33, 34) pour réguler un intervalle de rotation relatif dudit élément de commande (8, 41 ; 42) par rapport audit arbre de commande (5) et des parties en saillie en prise (31) ajustées respectivement entre lesdites parties en saillie et pouvant être en prise avec lesdites surfaces de régulation respectivement, et ledit dispositif de maintien de commande de rotation libre est pourvu de surfaces de régulation de rotation libre (53) positionnées hors de l'intervalle de régulation par chaque surface de régulation, de sorte que lorsque ladite poignée de manoeuvre (18) est mise dans la seconde position pour opérer une rota-

tion libre, lesdites parties en saillie en prise (31) sont adaptées pour venir respectivement en contact élastique avec lesdites surfaces de commande (53).

3. Engin de levage selon la revendication 2 dans lequel ledit dispositif de maintien de commande de rotation libre (31, 35 ; 53) est placé sur lesdites surfaces de commande de rotation libre et à l'avant dans la direction de fonctionnement de rotation libre de ladite poignée de manoeuvre avec des parties de régulation pour réguler l'intervalle de fonctionnement de rotation libre par ladite poignée de manoeuvre (18).
4. Engin de levage selon la revendication 1, dans lequel ledit arbre de commande (5) a une partie de positionnement pour régler la première position de ladite poignée de manoeuvre (18), la première position par ladite partie de positionnement étant mise dans une position où ladite poignée de manoeuvre n'est pas en contact avec ledit élément de commande (8, 41 ; 42).
5. Engin de levage selon la revendication 1 pourvu en outre d'un mécanisme de prévention de surcharge (40) comprenant :
ledit élément de commande comprenant un premier élément de commande (41) ayant une bosse pouvant être vissée avec ledit arbre de commande (5) et une partie de diamètre plus grande opposée à ladite plaquette de freinage sur ledit frein mécanique et un second élément de commande (42) maintenu de façon à pouvoir tourner avec un mouvement relatif, à ladite bosse dudit premier élément de commande, ladite bosse (41a) sur ledit premier élément de commande supportant des plaquettes de friction et un élément élastique et de vissage avec un élément de réglage (47) pour changer une force de tension desdites plaquettes de friction provoquée par ledit élément élastique afin de régler une charge de glissement, ledit élément de réglage (47) pourvu des parties de régulation pour réguler un intervalle de rotation relatif dudit élément de commande par rapport audit arbre de commande sur la première position de ladite poignée de manoeuvre et des surfaces de commande de rotation libre avec laquelle ladite poignée vient en contact élastique et pour maintenir la rotation libre dudit arbre de commande par ladite poignée de manoeuvre.
6. Engin de levage selon la revendication 5, dans lequel lesdites parties de régulation (52) dudit élément de réglage (45) comprennent chacune



une découpe ayant une paire de surfaces de régulation (54, 55) pour réguler l'intervalle de rotation relatif dudit élément de commande par rapport audit arbre de commande, ladite poignée de manoeuvre (18) étant pourvue des parties en saillie en prise (31) pour entrer dans lesdites découpes pour être en prise avec lesdites surfaces de régulation sur la première position de ladite poignée de manoeuvre et pour venir en contact élastique avec lesdites surfaces de commande de rotation libre sur la seconde position de ladite poignée de manoeuvre.

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7. Engin de levage selon la revendication 6, dans lequel ledit élément de réglage (47) est placé à l'avant de chacune desdites surfaces de commande de rotation libre et à l'avant dans la direction de manoeuvre de rotation libre de ladite poignée de manoeuvre avec des parties de régulation pour réguler un intervalle de fonctionnement de rotation libre par ladite poignée de manoeuvre.

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8. Engin de levage selon la revendication 5, dans lequel entre ladite bosse (41a) dudit premier élément de commande et dudit second élément de commande est fourni un mécanisme unidirectionnel de rotation (49) qui, lorsque ledit second élément de commande tourne dans la direction de rotation de commande, fait que ledit second élément de commande peut tourner librement par rapport audit premier élément de commande et, lorsque celui-ci tourne dans la direction de rotation non entraînée, tourne intégralement ledit second élément de commande avec ledit premier élément de commande, ledit mécanisme unidirectionnel de rotation comprenant un élément de prise (49) maintenu par un desdits premier et second éléments dans une relation pouvant être en saillie à partir de celui-ci ou rétractable à l'intérieur et une gorge en prise avec laquelle ledit élément en prise est en prise lorsque ledit second élément de commande tourne dans la direction de rotation non-entraînée, ladite gorge en prise étant fournie dans une pluralité et circonférentiellement.

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